















## 5. Coefficients and specimen's characteristics

### Compressive force

Load due to own weight 21.7 N

### Buckling rod

Length 2 × 250 mm

### Elastic joint

Tension spring constant  $k = 2 \text{ N/mm}$


Lever arm Length 50 mm

### Leaf spring:

Area moment of inertia  $6.66 \text{ mm}^4$

Material is steel with Elasticity modulus  $205000 \text{ N/mm}^2$

## 6. Safety Instructions

	<p><b>ATTENTION</b></p> <p>Respect the load mentioned in the procedure</p>
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## 7. Basic principles

The basic principles set out in the following make no claim to completeness. For further theoretical explanations, refer to the specialist literature.

A system is considered to be stable if it returns to its original position after being displaced from the equilibrium position, e.g. by an external force.

A particularly simple model for a system, which can be unstable, is the **buckling bar with elastic joint** (Fig. 3.2). This is stable until a certain loading condition. When this limit load (buckling force  $F_k$ ) is exceeded, where the system buckles suddenly, it will be unstable.

The elastic joint is facilitated by means of placing a turning spring between the two bar parts with torsional stiffness  $k_\varphi$ .



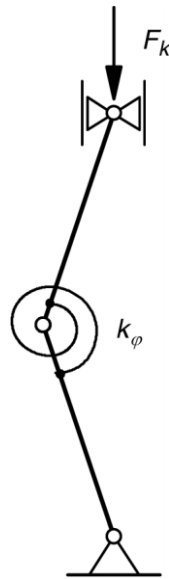


Fig. 3.2 Buckling bar with elastic joint

This spring will be realized in this training kit by using two tension springs (force constant  $k$ ) on two levers (length  $a$ ). First the torsional stiffness will be determined. Since the displacement is going to be small, the angle relation is linear:

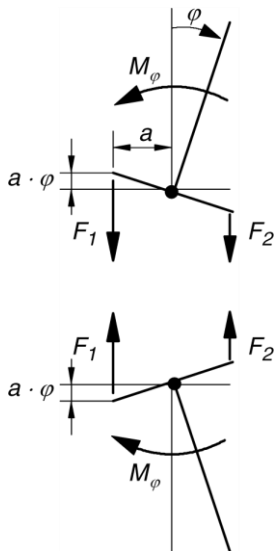


Fig. 3.3

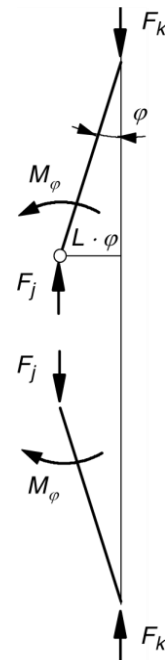


Fig. 3.4





$$\sum M_a = 0 = M_\varphi - F_j \cdot L \cdot \varphi = k_\varphi \cdot \varphi - F_j \cdot L \cdot \varphi \quad \text{Eq. 3.13}$$

At stability, this equilibrium condition must apply for non-zero displacement  $\varphi$  .

Equilibrium – upper bar

$$F_j \cdot L \cdot \varphi = F_Q \cdot L \quad \text{Eq. 3.14}$$

$$F_Q = F_j \cdot \varphi \quad \text{Eq. 3.15}$$

Equilibrium – lower bar

$$F_j \cdot L \cdot \varphi + F_j \cdot L \cdot \varphi = k_\varphi \cdot \varphi \quad \text{Eq. 3.16}$$

$$2 \cdot F_j \cdot L = k_\varphi \quad \text{Eq. 3.17}$$

$$F_j = \frac{k_\varphi}{2 \cdot L} \quad \text{Eq. 3.18}$$

The buckling force results from  $F_j = F_k$  and Formula (3.12) inserted in Formula (3.18)

$$F_k = \frac{3 \cdot E \cdot I}{2 \cdot b \cdot L} \quad \text{Eq. 3.19}$$

## 8. Experimental Procedure

### a. Experiment 1: Buckling with elastic joint

#### I. Experiment Aim

Determining the buckling force in the case of an elastic joint.

#### II. Preparing the Experiment

1. Clamp the upper bearing (9) on the lift vertical side of the frame. The height above the lower side of the frame is 600mm.
2. Clamp the lower joint bearing (8) straight below the hole of compression piece on the lower frame side.
3. Clamp the loading lever (5) with joint bearing (8) above the upper bearing (at 680mm height). The distance between the two bearings must be now 40mm.
4. Mount the compression piece (9a) in the upper bearing loosely. Insert the compression piece from the side with the clamping screws (9b)













