

Course Organization



Instructor

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Lectures

Wednesday 08:30 - 10:30 AM



Tutorials

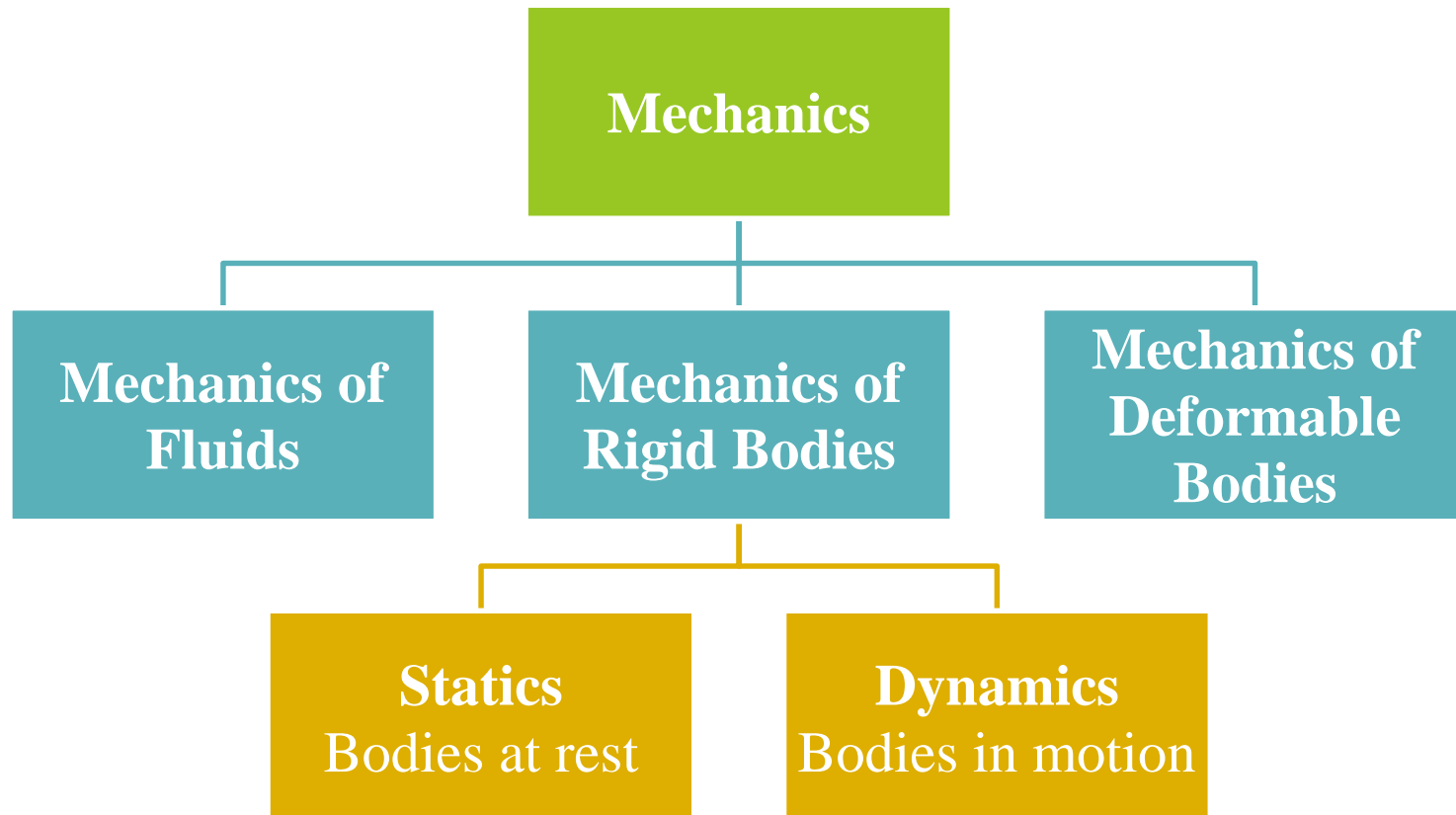
Gr A: Wednesday 16:00 - 18:00 PM

Gr B : Thursday 10:30 - 12:30 AM

Gr C: Wednesday 14:00 - 16:00 PM

Gr D: Thursday 08:30 - 10:30 AM

What is the Mechanics?

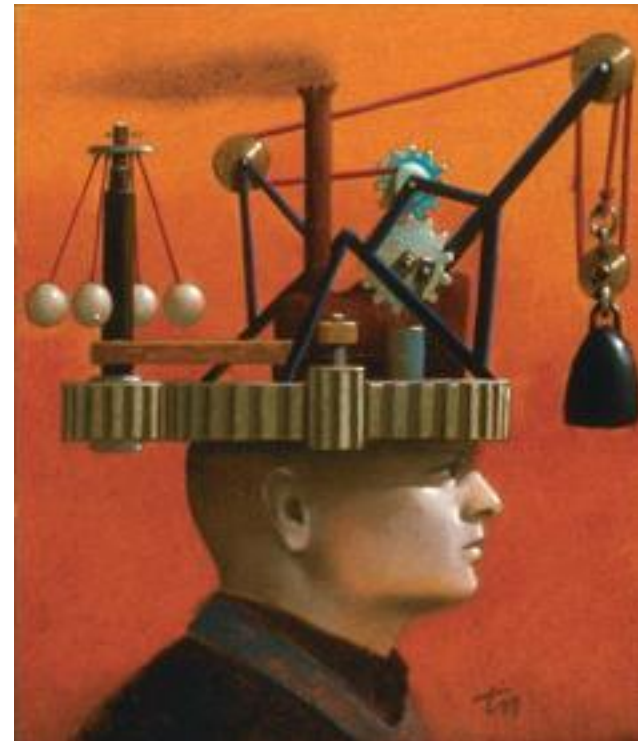


Mechanics of materials

- Studies the internal effects of stress and strain in a solid body that is subjected to an external loading.
- study of the body's stability when a body is subjected to defined loading

Problem-solving Procedures

For a given problem, three steps are used to solve the problems correctly:

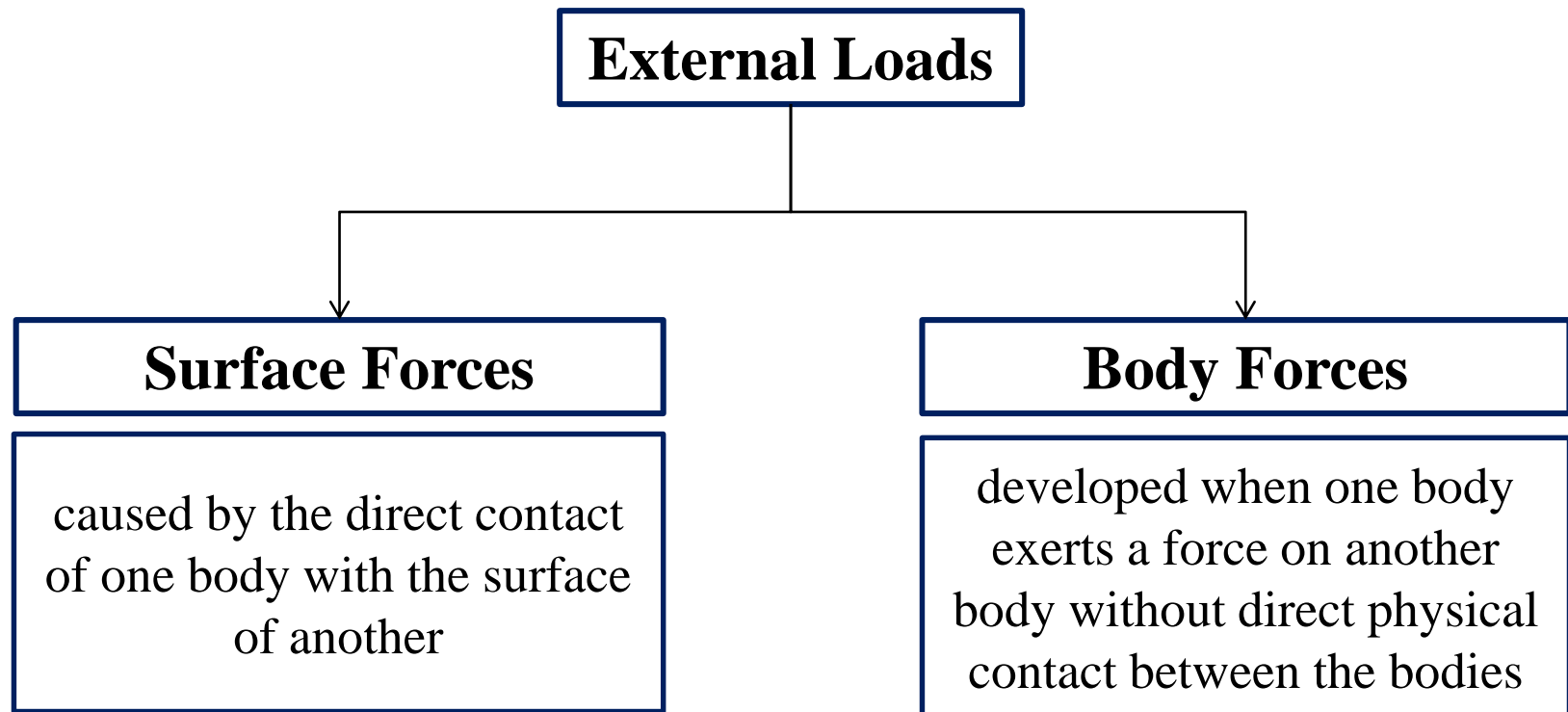


Problem-solving Procedures

For a given problem, three steps are used to solve the problems correctly:

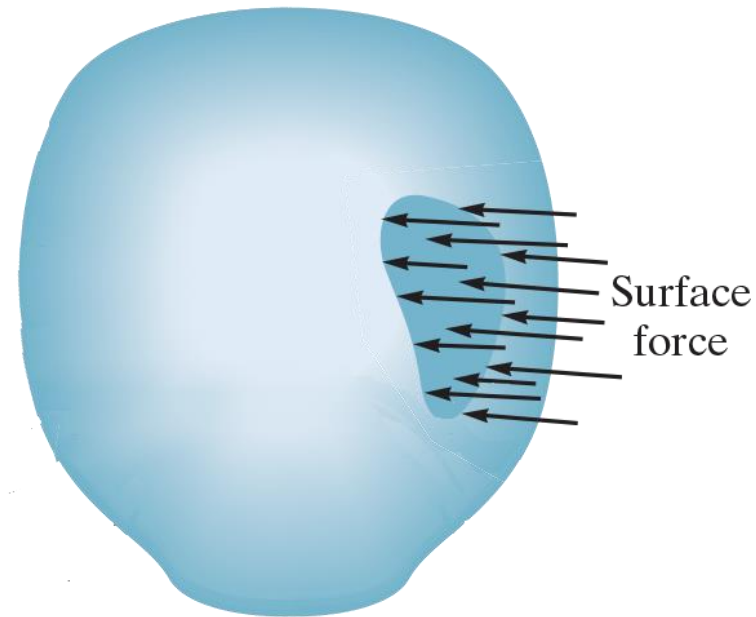
- A. Plan the solution: plan strategy and steps in solving the problems
- B. Carry out the solution: find the proper tools to solve the problems using equilibrium equations, geometry of deformation, and material type.
- C. Review the solution: Does the dimension make sense? Are the quantities in a reasonable manner including sign and magnitude? Does the solution violate the assumption you assumed before solving the problem?

External Loading



Surface forces

- ❑ Are caused by the direct contact of one body with the surface of another.
- ❑ Are distributed over the area of contact between the bodies.
- ❑ The loading is measured as having an intensity of force/surface over the area and is represented graphically by a series of arrows over the area.
- ❑ The resultant force : equivalent to $F = w(s).A$ and acts centroid C or geometric center



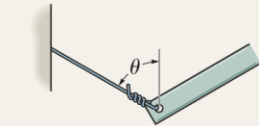
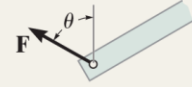

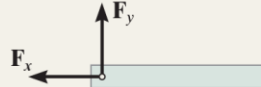

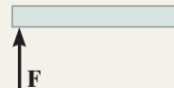

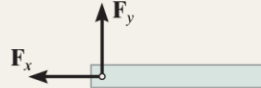



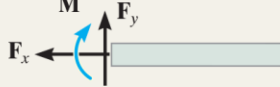
Support Reactions

The surface forces that develop at the supports or points of contact between bodies are called *reactions*.

If the support prevents translation in a given direction, then a force must be developed on the member in that direction. Likewise, if rotation is prevented, a couple moment must be exerted on the member

Support Reactions

The supports most commonly encountered for 2D problems, i.e., bodies subjected to coplanar force systems:

Type of connection	Reaction	Type of connection	Reaction
 <p>Cable</p>	 <p>One unknown: F</p>	 <p>External pin</p>	 <p>Two unknowns: F_x, F_y</p>
 <p>Roller</p>	 <p>One unknown: F</p>	 <p>Internal pin</p>	 <p>Two unknowns: F_x, F_y</p>
 <p>Smooth support</p>	 <p>One unknown: F</p>	 <p>Fixed support</p>	 <p>Three unknowns: F_x, F_y, M</p>

Note carefully the symbol used to represent each support and the type of reactions it exerts on its contacting member

Equilibrium of a Deformable Body

Equilibrium of a body requires both:

a ***balance of forces***, to prevent the body from translating or having accelerated motion along a straight or curved path, and
a ***balance of moments***, to prevent the body from rotating. These conditions can be expressed mathematically by two vector equations

$$\sum F = 0$$
$$\sum M_o = 0$$

Equilibrium of a Deformable Body

The force and moment vectors can be resolved into components along each coordinate axis and the two equations can be written in scalar form as six equations, namely

$$\sum F_x = 0, \sum F_y = 0, \sum F_z = 0$$
$$\sum M_x = 0, \sum M_y = 0, \sum M_z = 0$$

For planar system : for example in the plane xoy

$$\sum F_x = 0, \sum F_y = 0$$
$$\sum M_o = 0$$

Equilibrium of a Deformable Body

Successful application of the equations of equilibrium requires complete specification of all the known and unknown forces that act *on* the body, and so ***the best way to account for all these forces is to draw the body's free-body diagram.***

Procedure for Analysis

The resultant *internal* loadings at a point located on the section of a body can be obtained using the method of sections. This requires the following 3 steps:

1. Support Reactions.
2. Free-Body Diagram.
3. Equations of Equilibrium.

1- Support Reactions

- ✓ Decide which segment of the body is to be considered.
- ✓ If the segment has a support or connection to another body, then *before* the body is sectioned, it will be necessary to determine the reactions acting on the chosen segment.
- ✓ To do this draw the free-body diagram of the *entire body* and then apply the necessary equations of equilibrium to obtain these reactions.

2- Free-Body Diagram

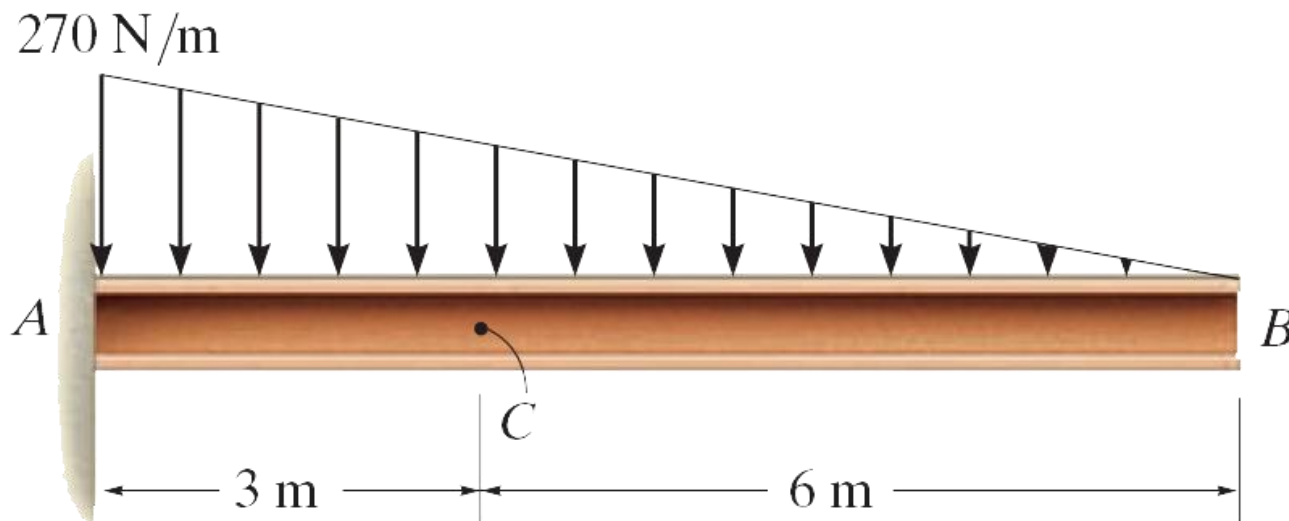
- ✓ Keep all external distributed loadings, couple moments, torques, and forces in their *exact locations*, before passing an imaginary section through the body at the point where the resultant internal loadings are to be determined.
- ✓ Draw a free-body diagram of one of the “cut” segments and indicate the unknown resultants **N**, **V**, **M**, and **T** at the section. These resultants are normally placed at the point representing the geometric center or *centroid* of the sectioned area.
- ✓ If the member is subjected to a *coplanar* system of forces, only **N**, **V**, and **M** act at the centroid.
- ✓ Establish the x , y , z coordinate axes with origin at the centroid and show the resultant internal loadings acting along the axes.

3- Equations of Equilibrium

- ✓ Moments should be summed at the section, about each of the coordinate axes where the resultants act. Doing this eliminates the unknown forces \mathbf{N} and \mathbf{V} and allows a direct solution for \mathbf{M} (and \mathbf{T}).
- ✓ If the solution of the equilibrium equations yields a negative value for a resultant, the assumed *directional sense* of the resultant is *opposite* to that shown on the free-body diagram.

Example 1

Determine the resultant internal loadings acting on the cross section at C of the cantilevered beam shown



Example 1

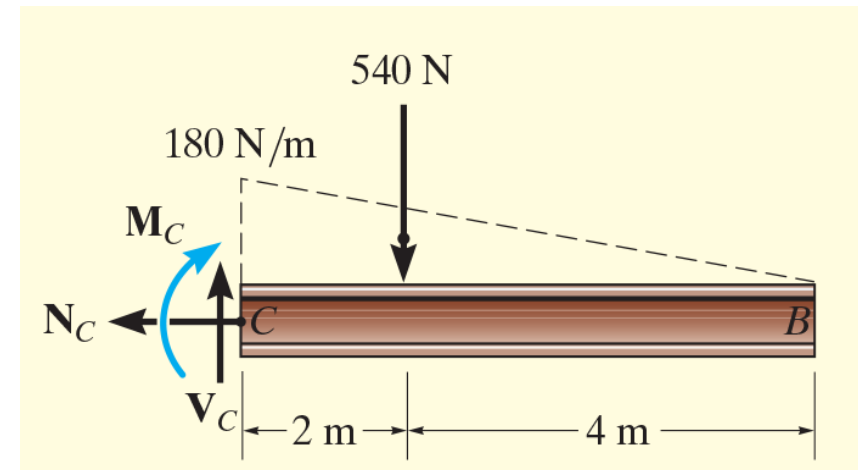
It is important to keep the distributed loading on the segment until *after* the section is made.

Only then should this loading be replaced by a single resultant force.

Notice that the intensity of the distributed loading at C is found by proportion.

$$\frac{W_1(s)}{L_1} = \frac{W_2(s)}{L_2} \Rightarrow \frac{270}{9} = \frac{W_c(s)}{6} \Rightarrow$$

$$W_c(s) = 6 \times 30 = 180 \text{ N/m}$$



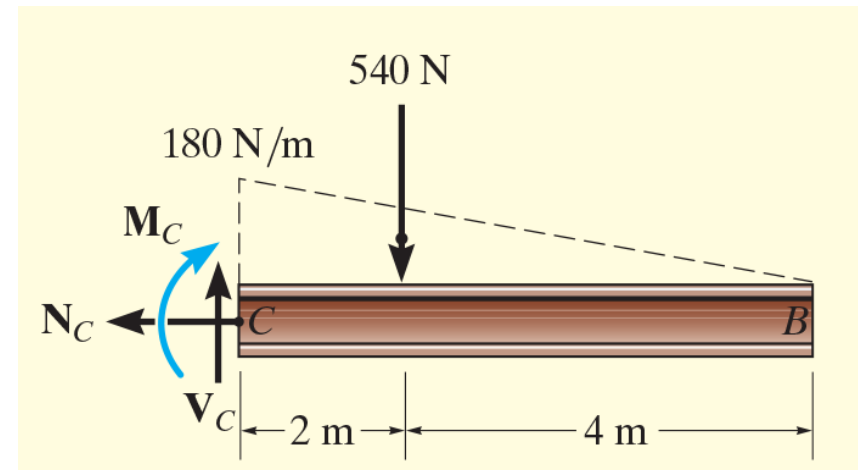
Example 1

The magnitude of the resultant of the distributed load is equal to the area under the loading curve (triangle) and acts through the centroid of this area.

$$F = \frac{W_c(s) \times CB}{2} = \frac{180 \times 6}{2} = 540 \text{ N}$$

Acts at : $\frac{1}{3}$ of CB

$$\Rightarrow \frac{1}{3} \times 6 = 2 \text{ m}$$



Example 1

3- Equations of Equilibrium.

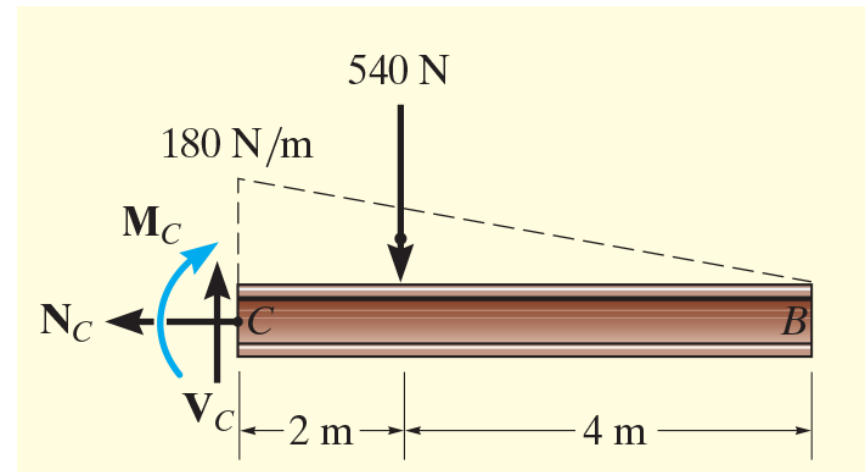
Applying the equations of equilibrium

$$\rightarrow \sum F_x = 0 \Rightarrow -N_c = 0 \Rightarrow N_c = 0 \quad N$$

$$\uparrow \sum F_y = 0 \Rightarrow V_c - 540N = 0 \Rightarrow V_c = 540 \quad N$$

$$\curvearrowleft \sum M_c = 0 \Rightarrow -M_c - 540N \times 2m = 0 \Rightarrow M_c = -1080 \quad N.m$$

The negative sign indicates that acts in the opposite direction to that shown on the free-body diagram.

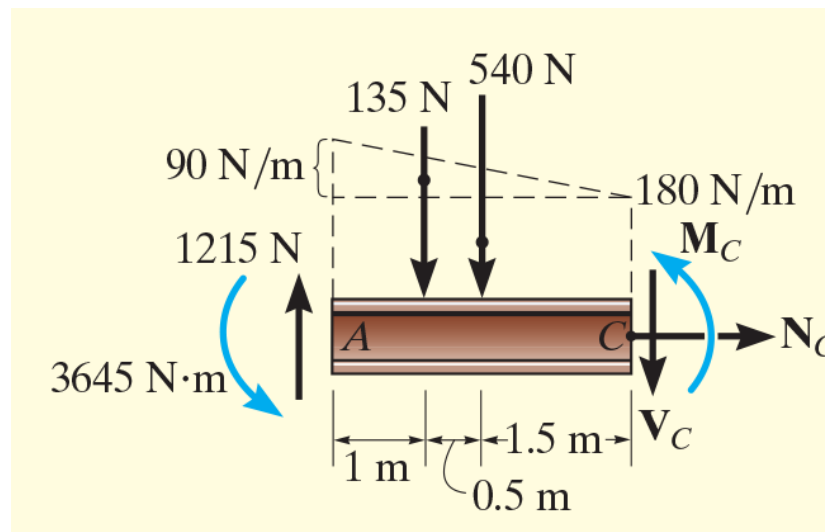


Example 1

The magnitude of the resultant of the distributed load is equal to the area under the loading curve (triangle + rectangle) and acts through the centroid of this area.

$$F_1 = \frac{W_{al}(s) \times AC}{2} = \frac{90 \times 3}{2} = 135 \text{ N acts at } \frac{1}{3} \text{ of AC} \Rightarrow \frac{1}{3} \times 3 = 1 \text{ m}$$

$$F_2 = W_{al}(s) \times AC = 180 \times 3 = 540 \text{ N acts at } \frac{1}{2} \text{ of AC} \Rightarrow \frac{1}{2} \times 3 = 1.5 \text{ m}$$



Example 1

3- Equations of Equilibrium.

Applying the equations of equilibrium we have

$$\begin{aligned} \rightarrow \sum F_x = 0 &\Rightarrow -N_c + N_A = 0 \\ &\Rightarrow N_c = N_A \end{aligned}$$

$$\begin{aligned} \uparrow \sum F_y = 0 &\Rightarrow V_A - 540N - 135N - V_C = 0 \\ &\Rightarrow V_A = V_C + 675N = 540N + 675N = 1215N \end{aligned}$$

$$\begin{aligned} \curvearrowright \sum M_c = 0 &\Rightarrow -M_c - 540N \times 1.5m - 135N \times 1m - 540N \times 3m + M_A = 0 \\ &\Rightarrow M_A = 945 + M_A N.m = 2565 + 1080 = 3645N.m \end{aligned}$$

